08/13/2019 SFF 2019 1400 – 1420 Hrs. Salon A

Experimental Validation of the Thermal Distribution Predicted by the Graph Theory Approach Application to Laser Powder Bed Fusion

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Outline

- Introduction
 - Research goal
 - Introduction of graph theoretic thermal modeling
- Experiment and simulation procedure
 - Experimental Setup
 - Build Strategies
- Result
 - Simulations vs. experiment
 - Graph Theory vs FEA
- Conclusion and Future Works
 - Distortion and microstructure prediction



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Validate the graph theory model using experimental laser powder bed fusion data

<u>Approach</u>

Yavari, M. R., Cole, K., and Rao, P., 2019, "Thermal Modeling in Metal Additive Manufacturing using Graph Theory," ASME Transactions, *Journal of Manufacturing Science and Engineering*, 141(7)

<u>Data</u>

Williams, R., Piglione, A., Ronneberg, T., Pham, M. S., Davis, C. M. and <u>Hooper, P. A.</u>, 2019, "In-situ thermography for laser powder bed fusion: effects of layer temperature on porosity, microstructure and mechanical properties", *Additive Manufacturing*, In Press



Graph Theoretic Thermal Modeling in AM





Recap: Verification with FEA

Graph theory simulates the AM process in C-shaped part.



Graph theory solution converges to similar trends as finite element analysis.

Yavari, M. R., Cole, K., and Rao, P., 2019, "Thermal Modeling in Metal Additive Manufacturing using Graph Theory," ASME Transactions, *Journal of Manufacturing Science and Engineering*, 141(7), pp. 071007-071027



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A thermal camera is used to measure the surface temperature on the top surface.



point distance: 40 μ m, exposure time: 50 μ s



- Calibration function applied to convert the raw IR camera data to temperature values.
- IR camera was calibrated empirically for both solid and powder.
- AM part temperature was controlled using a cartridge heater.
- Absolute temperature trends captured using thermocouples embedded in a cavity.



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Build 1 (Cylinder)

Build 1 includes of 3 different phases:

- Phase 1: Print 9 cylinders (dia. 8 mm, L = 60 mm).
- Phase 2: Print only the middle cylinder.

Phase 3: Print all 9 cylinders again.



Change in the build plan causes variation in the inter-layer cooling time (ILCT).



The temperature recorded at center pixel of the middle cylinder.



Thermal data need to be filtered to remove IR transients.

Raw IR camera measurements includes of several high and low peaks.







Removing Transients from IR Data

1 : Laser is located at the pixel traced by the IR-camera images





Inter-layer cooling time (ILCT): The time between successive scans, layer-to-layer.



- Phase 1: 10 seconds
- Phase 2: 6 seconds
- Phase 3: 10 seconds



Removing Transients from IR Data

2 : Recoater returns to back for powder





Removing Transients from IR Data

③ : Recoater spreads powder for a new layer





Top surface temperature for 1200 layers.







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Graph Theoretic Thermal Modeling in AM



Step 1- Convert the part into a set of discrete nodes



Step 3

- Deposition of super layers
- Diffusion of the heat through the part



Step 2- Network graph construction



Step 4- Obtaining the result



Super layer thickness = 3 mm (60 actual layers) Computational time= 75 seconds ~ (1 minutes) Mean Absolute Percentage Error (MAPE) = 29 %





Super layer thickness = 2 mm (40 actual layers) Computational time= 166 seconds ~ (3 minutes) Mean Absolute Percentage Error (MAPE) = 24 %





Super layer thickness = 1 mm (20 actual layers) Computational time= 481 seconds ~ (8 minutes) Mean Absolute Percentage Error (MAPE) = 16 %





Super layer thickness = 0.3 mm (6 actual layers) Computational time= 1,655 seconds ~ (27 minutes) Mean Absolute Percentage Error (MAPE) = 6 %





Number of Nodes= 1,000

Computational time= 92 seconds ~ (1 minutes)

Mean Absolute Percentage Error (MAPE) = 42 %





Number of Nodes= 2,000

Computational time= 501 seconds \sim (8 minutes) Mean Absolute Percentage Error (MAPE) = 15 %





Number of Nodes= 3,000

Computational time= 1,655 seconds \sim (27 minutes) Mean Absolute Percentage Error (MAPE) = 6 %





Effect of number of nodes and super layer thickness on accuracy and computation time. (Simulation Time/Build Time)*100 MAPE (%) 0.3 Number of Nodes 0.3^{0.4}^{0.5}^{0.6}^{0.8}¹²³ Super Layer Thickness 0.5 Super Layer Thickness Number of Nodes

Increasing number of nodes and decreasing super layer thickness reduces error and inflates computation time.



Graph theory approach and FE approach were compared.



| Actual Build Time 171 minutes | Finite Element | | Graph Theory | |
|-------------------------------|----------------|------------|--------------|------------|
| Super Layer Thickness | 0.3 mm | 0.5 mm | 0.3 mm | 0.5 mm |
| Computation Time | 34 minutes | 22 minutes | 27 minutes | 15 minutes |
| MAPE | 8 % | 18 % | 6 % | 14 % |
| RMSE (Kelvin, K) | 33.8 | 48.1 | 14.5 | 33.8 |

Graph theory converges faster than FE, and slightly smaller error.



Graph theory approach and FE approach were compared.



- Both the FE model and graph theory approach use super layer.
- Number of nodes are similar (N = 4000)
- Graph theory algorithm has not yet optimized for parallel processing
- Algorithm is currently implemented in a derived computation language Matlab (single core processing)



Same calibration methods and temperature data characterization.



180,000 frames, 60 frames per second.



316L Stainless Steel Build time: 51 minutes Laser power: 200 W



Super layer thickness = 1 mm (20 actual layers) Computational time= 237 seconds ~ (4 minutes) Mean Absolute Percentage Error (MAPE) = 43 %





Super layer thickness = 0.8 mm (16 actual layers) Computational time= 721 seconds ~ (12 minutes) Mean Absolute Percentage Error (MAPE) = 32 %





Effect of Super Layer Thickness

Super layer thickness = 0.2 mm (4 actual layers) Computational time= 2,471 seconds ~ (41 minutes) Mean Absolute Percentage Error (MAPE) = 8 %





Number of Nodes= 1,000

Computational time= 104 seconds ~ (2 minutes) Mean Absolute Percentage Error (MAPE) = 34 %





Number of Nodes= 3,000

Computational time= 1,017 seconds \sim (17 minutes) Mean Absolute Percentage Error (MAPE) = 20 %





Number of Nodes= 4,000

Computational time= 2,471 seconds ~ (41 minutes) Mean Absolute Percentage Error (MAPE) = 8 %





Effect of number of nodes and super layer thickness on accuracy and computation time.





Table and plot shows the finite element and graph theory results comparison

| Actual Build Time 53 minutes | Finite Element | | Graph Theory | |
|------------------------------|----------------|------------|--------------|------------|
| Super Layer Thickness | 0.2 mm | 0.3 mm | 0.2 mm | 0.3 mm |
| Computation Time | 54 minutes | 48 minutes | 41 minutes | 35 minutes |
| MAPE | 9 % | 14 % | 8 % | 9 % |
| RMSE (Kelvin, K) | 37.7 | 73.0 | 26.0 | 35.4 |





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- Graph theoretic simulation is able to simulate the top surface temperature of the part in Laser Powder Bed Fusion process.
- Use graph theoretic thermal filed to predict part distortion.
- Use graph theoretic thermal filed to predict microstructure.







Advanced Manufacturing Processes and Sensing

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Comparison with Exact Analytical Solution

Purpose: Quantify the accuracy of graph theory diffusion with analytical solution

Geometry condition: (W = L = H = 1) and ($W_1 = L_1 = H_1 = 0.5$)

Observation points: Point 1 = (0.25H, 0.25L, 0.25W), Point 2 = (0.75H, 0.75L, 0.75W).



Cole et al., 2011, chap. 3. Heat Conduction Using Green's Functions, Exact Analytical Conduction Toolbox (EXACT) at UNL www. exact.unl.edu

Comparison with Exact Analytical Solution

Graph theory captures the physics of the heat transfer for an ideal case.



| Error | Graph theoretic approach time (sec.) | | FE anal | FE analysis time (sec.) | | |
|-------|--------------------------------------|---|---------|-------------------------|--|--|
| ~ 5% | 237 | | | 3,540 | | |
| | 4 mins | 5 | 59 mins | | | |

The Scientific Problem

Temperature (T) is a function of space (x, y, z) and time (t)

T(*x*, *y*, *z*, *t*)



The Heat Equation (Fourier's Law of Conduction)

$$\rho c_p \frac{\partial T(x, y, z, t)}{\partial t} - k \left(\frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2} + \frac{\partial^2}{\partial z^2} \right) T(x, y, z, t) = 0$$
$$\rho c_p \frac{\partial T}{\partial t} - k \left(\frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2} + \frac{\partial^2}{\partial z^2} \right) T = 0$$

K = thermal conductivity ρ = density C_p = specific heat

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Solving the Heat Equation with Graph Theory

$$\frac{\partial T}{\partial t} - \frac{k}{\rho c_p} \left(\frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2} + \frac{\partial^2}{\partial z^2} \right) T = 0$$

Laplacian operator

$$\Delta \stackrel{\text{\tiny def}}{=} \left(\frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2} + \frac{\partial^2}{\partial z^2} \right)$$

Continuum heat equation $\frac{\partial T}{\partial t} - \alpha(\Delta)T = 0$

 $k/\rho c_p = \alpha$ (Thermal diffusivity)

Hypothesis

The Heat Equation is solved as a function of the Eigenvalues (Λ) and Eigenvectors (ϕ) of the Discrete Laplacian Matrix (\mathcal{L})

$$\frac{\partial \mathbf{T}}{\partial t} - \alpha(\mathbf{\Delta})\mathbf{T} = \mathbf{0}$$

The continuous Laplacian operator is approximated by the Graph Laplacian.

 $\Delta \approx -\mathcal{L}$

Describing the Laplacian matrix by its eigenspectrum:

$$\mathcal{L} = \varphi \lambda^* \varphi^{-1}$$

$$\mathbf{T} = e^{-\alpha \mathbf{g}(\mathbf{\phi} \mathbf{\Lambda} \mathbf{\phi'})t}$$



Find the Gaussian distance between nodes (Closer nodes have higher edge weights)

$$w_{ij} = e^{-\frac{\left(\overrightarrow{x_i} - \overrightarrow{x_j}\right)\left(\overrightarrow{x_i} - \overrightarrow{x_j}\right)^{\mathrm{T}}}{\sigma^2}}$$

Similarity matrix
$$S^{M \times M} \stackrel{\text{\tiny def}}{=} [w_{ij}]$$



Obtaining Eigenvectors (ϕ) and Eigenvalues (Λ) 50



Degree matrix

$$\mathcal{D} \stackrel{\text{\tiny def}}{=} \begin{bmatrix} d_1 & 0 & 0 \\ 0 & d_k & 0 \\ 0 & 0 & d_M \end{bmatrix}$$

Matrix of Real positive numbers

Laplacian matrix

 $\mathcal{L} \stackrel{\text{\tiny def}}{=} (\mathcal{D} - S)$

$$\mathcal{L}\phi = \Lambda\phi$$



- Test has been done for Phase 1 of the cylinder (Build 1).
- Best gain factor calculated and used for the the Build 1 and 2.



